

# MSE 222 DYNAMICS Final Exam

SIMON FRASER UNIVERSITY  
MECHATRONIC SYSTEMS ENGINEERING

Final Examination – April 17, 2016

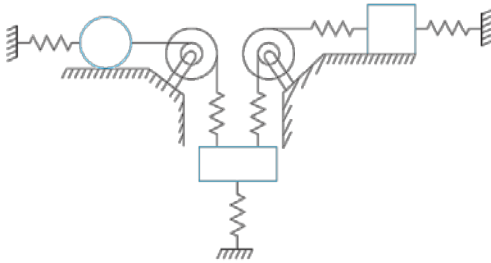
Instructor: Kambiz Hajikolaie

Time: 150 minutes

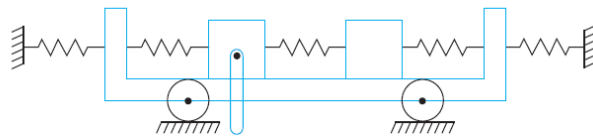
Non-programmable calculators may be used  
No smartphones or other electronic devices may be used  
Answer all the questions in the booklet (not the question sheet)

## Section I: short answer questions (10 marks)

Q1. Determine the number of degrees of freedom in the following dynamic systems:



(a)



(b)

Q2. Define the following terms in a single-degree-of-freedom vibrational system:

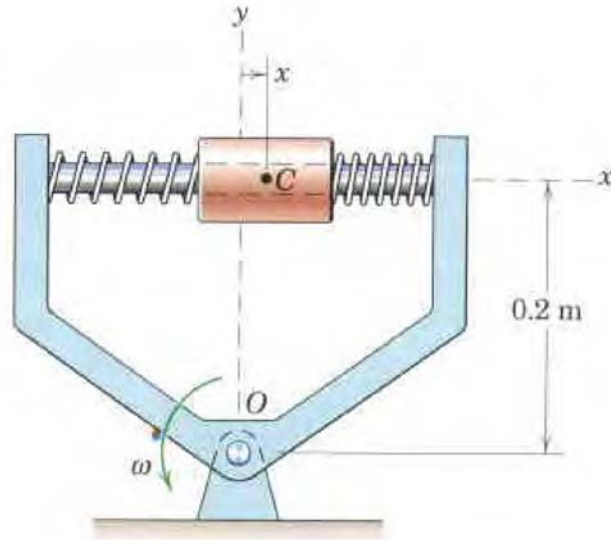
- Period of oscillation
- Amplitude
- Phase angle
- Critically damped motion

Q3. True-False questions:

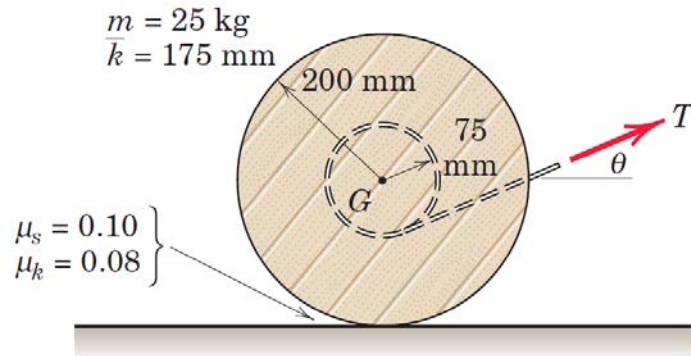
- Free vibration takes place at the system's natural frequency, irrespective of the initial conditions.
- Natural frequencies depend only on the stiffness properties of the system.
- Natural frequency decreases with an increase in the stiffness or a decrease in the mass.
- When an underdamped system is disturbed from rest, the motion is non-periodic.

**Section II: Problems**

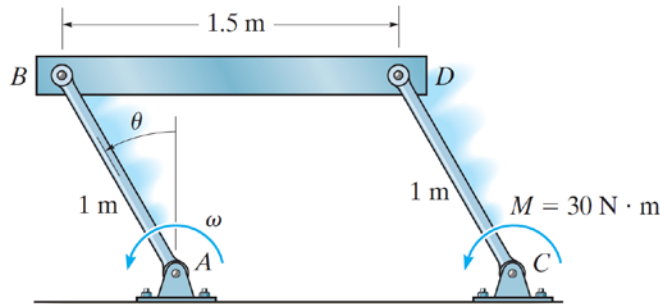
**Problem1:** The spring-mounted collar oscillates on the shaft according to  $x = 0.04 \sin(\pi t)$ , where  $x$  is in meters and  $t$  is in seconds. Simultaneously the frame rotates about the bearing at  $O$  with an angular velocity  $\omega = 2 \sin\left(\frac{\pi t}{2}\right)$  rad/s. Determine the acceleration of the center  $C$  of the collar (a) when  $t=3s$  and (b) when  $t=0.5s$ . (25 marks)



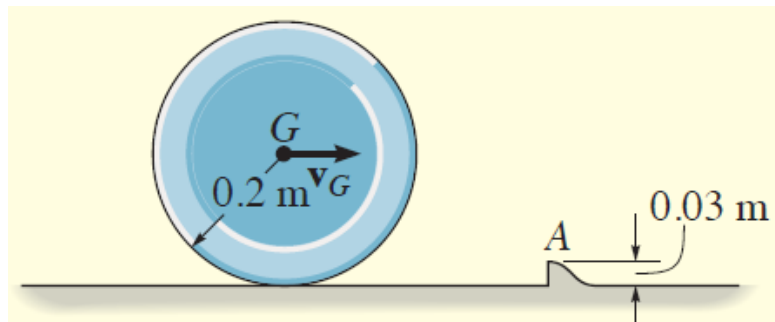
**Problem2:** The circular disk of 200-mm radius has a mass of 25 kg with centroidal radius of gyration  $k = 175 \text{ mm}$  and has a concentric circular groove of 75 mm radius cut into it. A steady force  $T$  is applied at an angle  $\theta$  to a cord wrapped around the groove as shown. If  $T = 30 \text{ N}$ ,  $\theta = 0$ ,  $\mu_s = 0.1$  and  $\mu_k = 0.08$ , determine the angular acceleration  $\alpha$  of the disk, the acceleration  $a$  of its mass center  $G$ , and the friction force  $F$  which the surface exerts on the disk. (20 marks)



**Problem3:** The linkage consists of two 6-kg rods AB and CD and a 20-kg bar BD. When  $\theta = 0^\circ$ , rod AB is rotating with an angular velocity  $\omega = 2 \text{ rad/s}$ . If rod CD is subjected to a couple moment of  $M = 30 \text{ N}\cdot\text{m}$ , determine  $\omega_{AB}$  at the instant  $\theta = 90^\circ$ . (20 marks)



**Problem4:** The 10-kg wheel shown in the figure has a moment of inertia  $I_G = 0.156 \text{ kg}\cdot\text{m}^2$ . Assuming that the wheel does not slip or rebound, determine the minimum velocity  $v_G$  it must have to just roll over the obstruction at A. (25 marks)



**Kinematics Formulas:**

Rigid-body analysis:

$$\mathbf{v}_B = \mathbf{v}_A + \boldsymbol{\omega} \times \mathbf{r}_{B/A} \qquad \mathbf{a}_B = \mathbf{a}_A + \boldsymbol{\alpha} \times \mathbf{r}_{B/A} - \omega^2 \mathbf{r}_{B/A}$$

Relative motion:

$$\mathbf{v}_B = \mathbf{v}_A + \boldsymbol{\Omega} \times \mathbf{r}_{B/A} + (\mathbf{v}_{B/A})_{xyz}$$

$$\mathbf{a}_B = \mathbf{a}_A + \dot{\boldsymbol{\Omega}} \times \mathbf{r}_{B/A} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}_{B/A}) + 2\boldsymbol{\Omega} \times (\mathbf{v}_{B/A})_{xyz} + (\mathbf{a}_{B/A})_{xyz}$$

**Moment of Inertia Formulas:**

General Formula

$$I = \int_m r^2 \, dm$$

Parallel-axis theorem

$$I = I_G + md^2$$

**Kinetics Formulas:**

$$\Sigma F_x = m(a_G)_x \quad \Sigma F_y = m(a_G)_y \quad \Sigma M_G = 0$$

Rectilinear translation

$$\Sigma F_n = m(a_G)_n \quad \Sigma F_t = m(a_G)_t \quad \Sigma M_G = 0$$

Curvilinear translation

$$\Sigma F_n = m(a_G)_n = m\omega^2 r_G$$

$$\Sigma F_t = m(a_G)_t = m\alpha r_G$$

$$\Sigma M_G = I_G \alpha \text{ or } \Sigma M_O = I_O \alpha$$

Rotation about a fixed axis

$$\Sigma F_x = m(a_G)_x$$

$$\Sigma F_y = m(a_G)_y$$

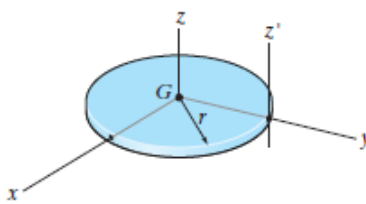
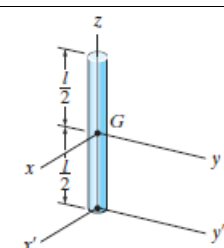
$$\Sigma M_G = I_G \alpha \text{ or } \Sigma M_P = \Sigma (\mathcal{M}_k)_P$$

General plane motion

<b>Work and Energy Formulas:</b>	
Kinetic Energy (Translation): $T = \frac{1}{2}mv_G^2$	
Kinetic Energy(General Plane Motion): $T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G\omega^2$ or $\frac{1}{2}I_C\omega^2$	Kinetic Energy (Rotation about a fixed axis): $T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G\omega^2$ or $\frac{1}{2}I_O\omega^2$
Principle of work and energy: $T_1 + \Sigma U_{1-2} = T_2$	Conservation of energy: $T_1 + V_1 = T_2 + V_2$ where $V = V_g + V_e$

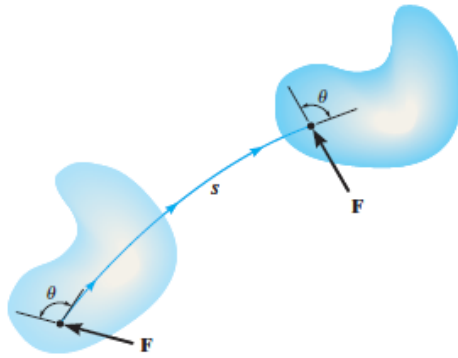
<b>Impulse and Momentum Formulas:</b>		
Linear and Angular Momentum:		
Translation $L = mv_G$ $H_G = 0$ $H_A = (mv_G)d$	Rotation about a fixed axis $L = mv_G$ $H_G = I_G\omega$ $H_O = I_O\omega$	General plane motion $L = mv_G$ $H_G = I_G\omega$ $H_A = I_G\omega + (mv_G)d$
Principle of Impulse and Momentum $m(v_{Gx})_1 + \Sigma \int_{t_1}^{t_2} F_x dt = m(v_{Gx})_2$ $m(v_{Gy})_1 + \Sigma \int_{t_1}^{t_2} F_y dt = m(v_{Gy})_2$ $I_G\omega_1 + \Sigma \int_{t_1}^{t_2} M_G dt = I_G\omega_2$	Conservation of Momentum $\left( \Sigma \text{ syst. linear momentum} \right)_1 = \left( \Sigma \text{ syst. linear momentum} \right)_2$ $\left( \Sigma \text{ syst. angular momentum} \right)_{O1} = \left( \Sigma \text{ syst. angular momentum} \right)_{O2}$	
Coefficient of Restitution: $e = \frac{(v_B)_2 - (v_A)_2}{(v_A)_1 - (v_B)_1}$		

### Center of gravity and mass moment of inertia of homogeneous solids

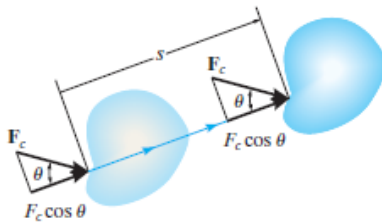
 <p>Thin Circular disk</p> $I_{xx} = I_{yy} = \frac{1}{4}mr^2 \quad I_{zz} = \frac{1}{2}mr^2 \quad I_{z'z'} = \frac{3}{2}mr^2$	 <p>Slender Rod</p> $I_{xx} = I_{yy} = \frac{1}{12}ml^2 \quad I_{x'x'} = I_{y'y'} = \frac{1}{3}ml^2 \quad I_{z'z'} = 0$
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**Work of a Force and a Couple Moment**

A force does work when it undergoes a displacement  $ds$  in the direction of the force. In particular, the frictional and normal forces that act on a cylinder or any circular body that rolls *without slipping* will do no work, since the normal force does not undergo a displacement and the frictional force acts on successive points on the surface of the body.

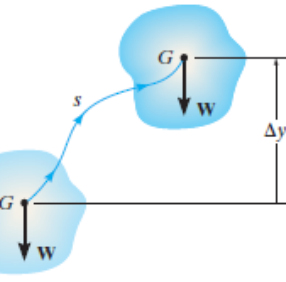


$$U_F = \int F \cos \theta ds$$



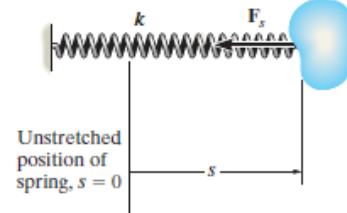
$$U_{F_c} = (F_c \cos \theta)s$$

Constant Force



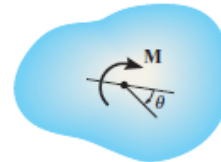
$$U_w = -W\Delta y$$

Weight



$$U = \frac{1}{2}k s^2$$

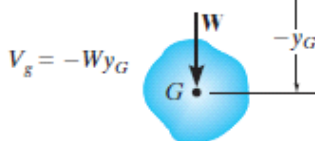
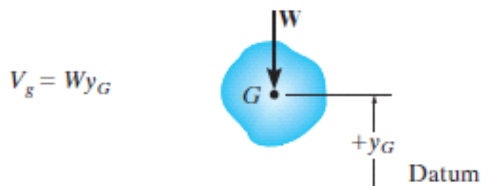
Spring



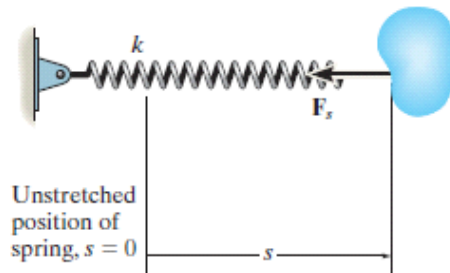
$$U_M = \int_{\theta_1}^{\theta_2} M d\theta$$

$$U_M = M(\theta_2 - \theta_1)$$

Constant Magnitude



Gravitational potential energy



$$V_e = \frac{1}{2}ks^2$$

Elastic potential energy